

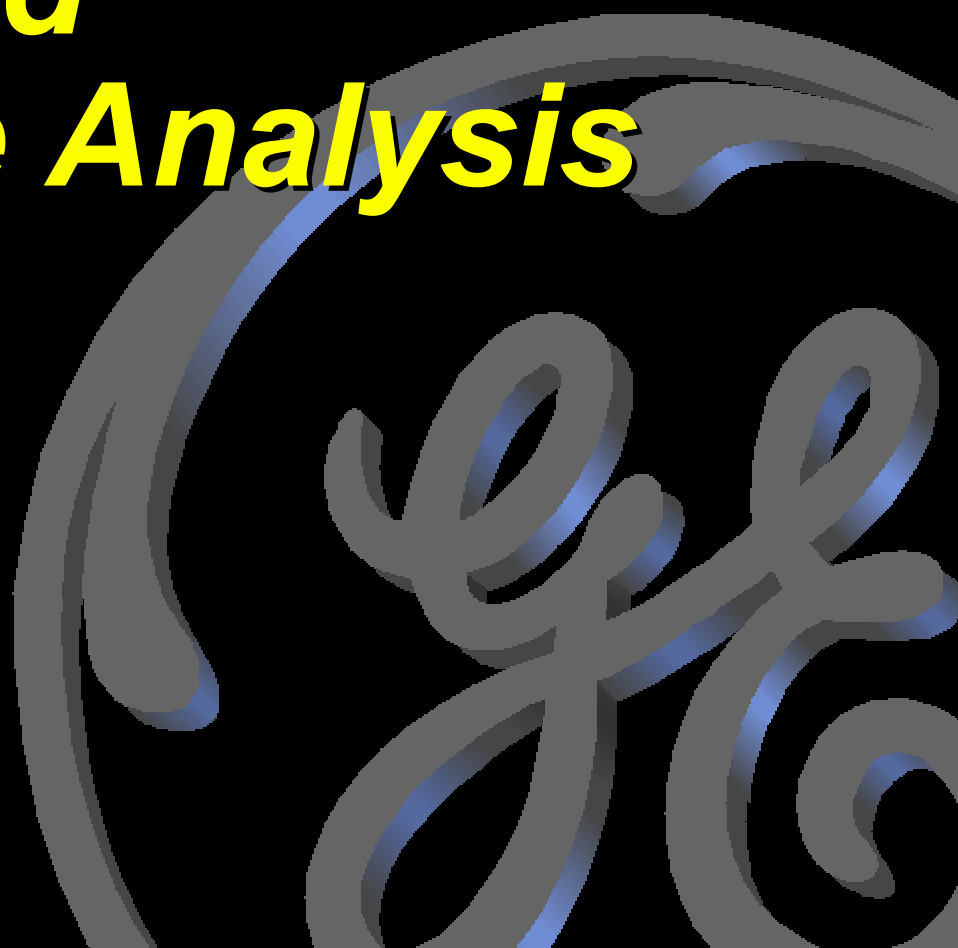


***GE Global Research***

# ***Exergy Based Microturbine Analysis***

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# Outline

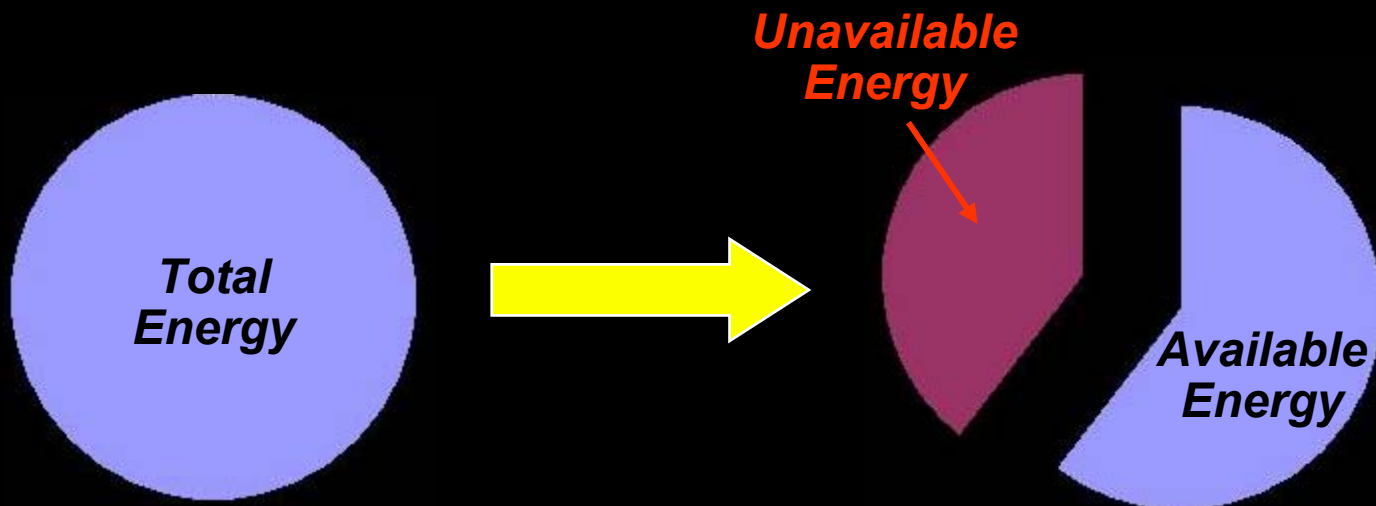
- What is Exergy ?
- First Law Versus Second Law
- Second Law Application – Air Standard Brayton Cycle
- Chemical Availability And Combustion Losses
- Summary





# What is Exergy ?

- *Exergy Represents the Maximum Work Potential A Device Can Deliver*
- Term Made Popular in Europe in 1950's
- Synonymous with "Availability" (Term introduced by M.I.T. in 1940's)
- Unlike Energy, Exergy Can Be Destroyed
  - ✓ Friction
  - ✓ Uncontrolled Expansion/Compression
  - ✓ Heat Transfer
- Allows For Greater Understanding of Component and System Efficiencies





# ***First Law of Thermodynamics***

$$M_{\text{air\_earth}} = 1.03 \times 10^{19} \text{ lbm}$$

$$T_0 = 59 \text{ }^{\circ}\text{F}$$

$$E_{\text{air\_earth}} = 1.278 \times 10^{21} \text{ BTU}$$

$$t = 1.0 \times 10^6 \text{ years}$$

$$\eta = 0.01$$

$$P_{\text{air\_earth}} = (E_{\text{air\_earth}} \times \eta) / t$$

$$P_{\text{air\_earth}} = 427.4 \text{ MW}$$



The First Law of Thermodynamics deals with the *quantity* of energy and asserts that energy cannot be created or destroyed



# Second Law of Thermodynamics

$$M_{\text{air\_earth}} = 1.03 \times 10^{19} \text{ lbm}$$

$$T_0 = 59^\circ \text{F}$$

$$E_{\text{air\_earth}} = 1.278 \times 10^{21} \text{ BTU}$$

$$t = 1.0 \times 10^6 \text{ years}$$

$$\eta = 0.01$$

$$W_{\text{air\_earth}} = (1 - T_{\text{sink}}/T_{\text{source}}) \times Q_{\text{in}}$$

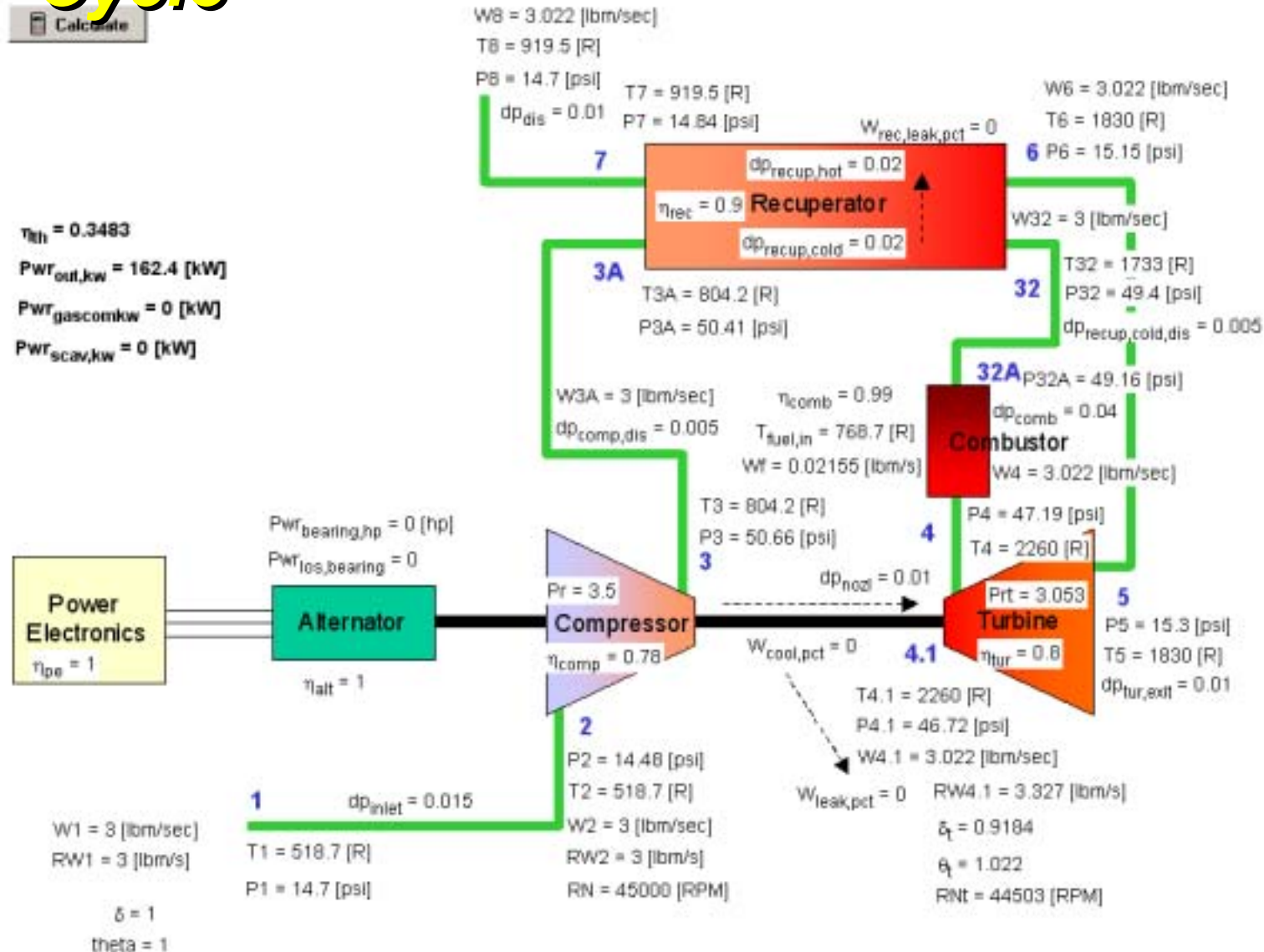
$$W_{\text{air\_earth}} = 0$$



The Second Law of Thermodynamics deals with the *quality* of energy and the degradation of energy during a process



# Recuperated Air Standard Brayton Cycle





# First And Second Law Work Together

$$q_{\text{avail\_out}} = (h_8 - h_1) - (s_8 - s_1) \cdot T_o$$

$$\eta_{\text{rec}} = \frac{h_{32} - h_{3a}}{h_6 - h_{3a}}$$

$$\eta_{\text{th}} = \frac{P_{\text{out}}}{W_f \cdot \text{LHV}_{\text{CH}_4}}$$

$$W_6 \cdot h_6 + W_{3a} \cdot h_{3a} = W_{32} \cdot h_{32} + W_7 \cdot h_7 + Q$$

$$W_{\text{av}} = Q_{\text{in\_total}} - q_{\text{avail\_out}} - \Delta S_{\text{total}} \cdot T_o$$

$$\eta_s = \frac{W_{\text{av}}}{Q_{\text{in\_total}}}$$

$$T_3 - T_2 = \frac{T_2}{\eta_{\text{comp}}} \cdot \left[ \text{Pr}^{\frac{(\gamma_2 - 1)}{\gamma_2}} - 1 \right]$$

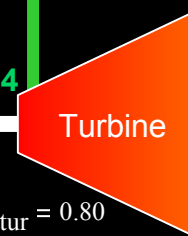
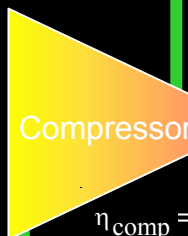
$$W_{\text{comp}} = m \cdot (h_3 - h_2)$$

$$\text{FAR}_{\text{ac}} = \frac{\text{FAR}_{\text{th}}}{\eta_{\text{comb}}}$$

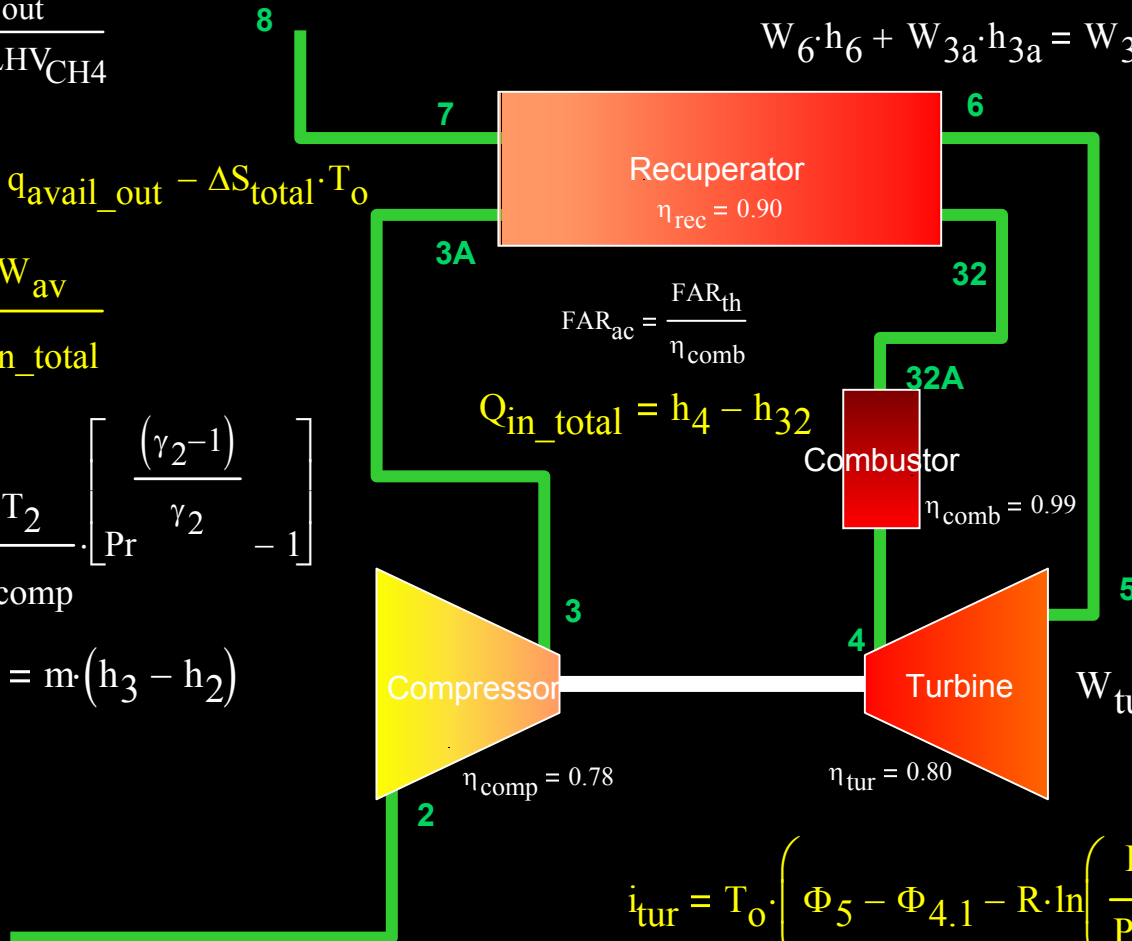
$$Q_{\text{in\_total}} = h_4 - h_{32}$$

$$\Delta S_{\text{total}} = \sum \Delta S_i$$

Combustor  
 $\eta_{\text{comb}} = 0.99$



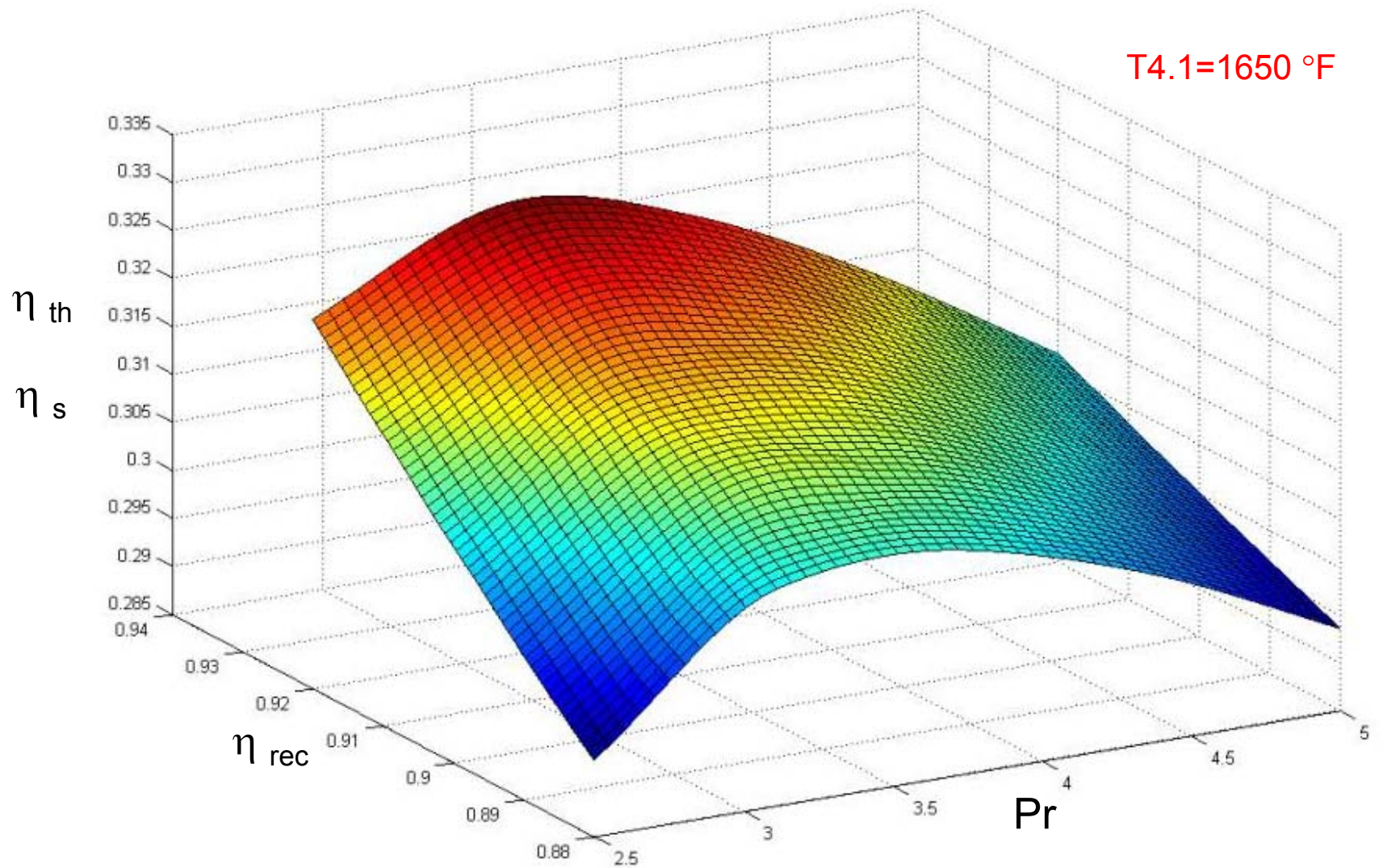
$$i_{\text{tur}} = T_o \cdot \left( \Phi_5 - \Phi_{4.1} - R \cdot \ln \left( \frac{P_5}{P_{4.1}} \right) \right) = T_o \cdot (s_5 - s_{4.1})$$







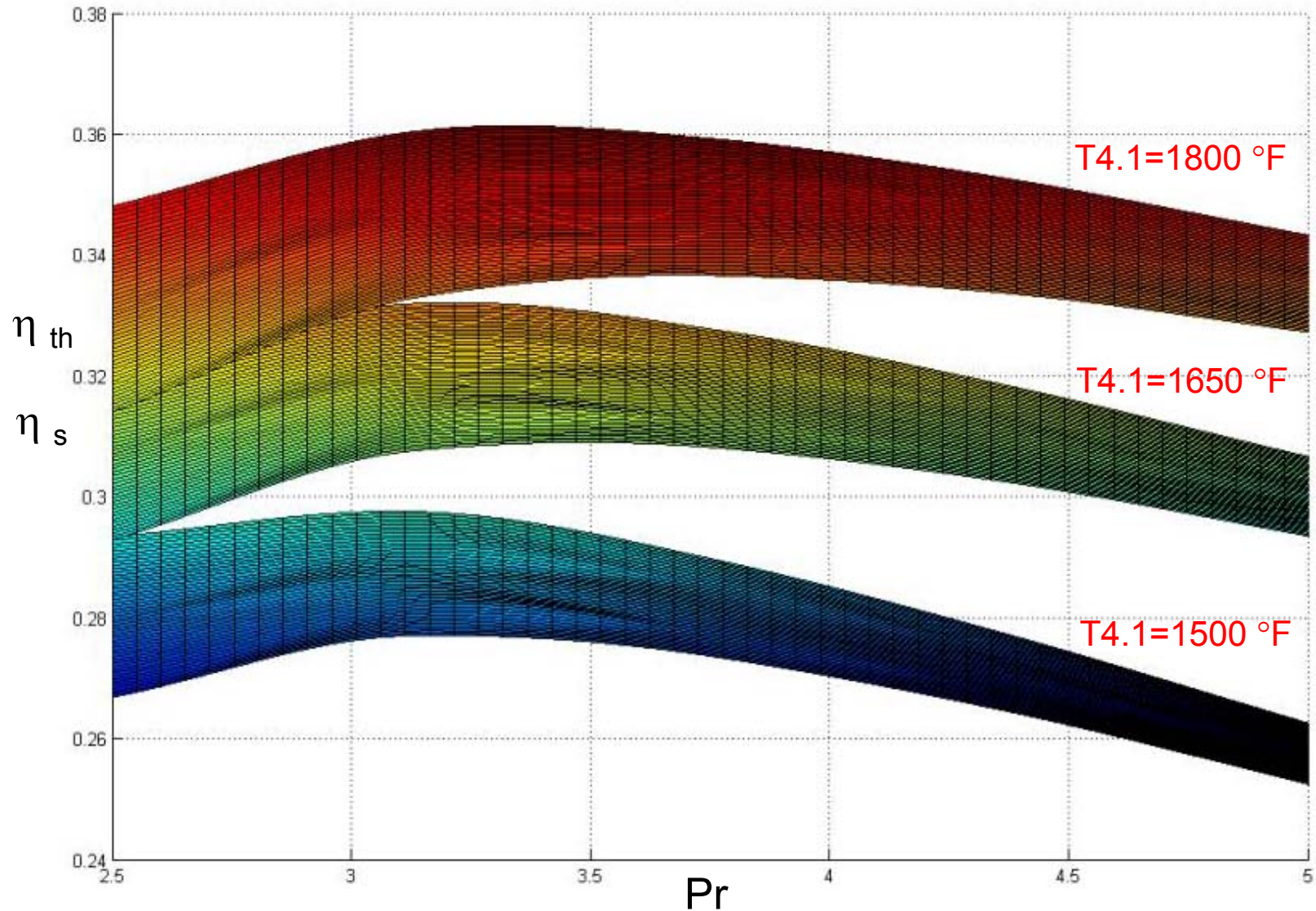
# Agreement Between First and Second Law Analysis





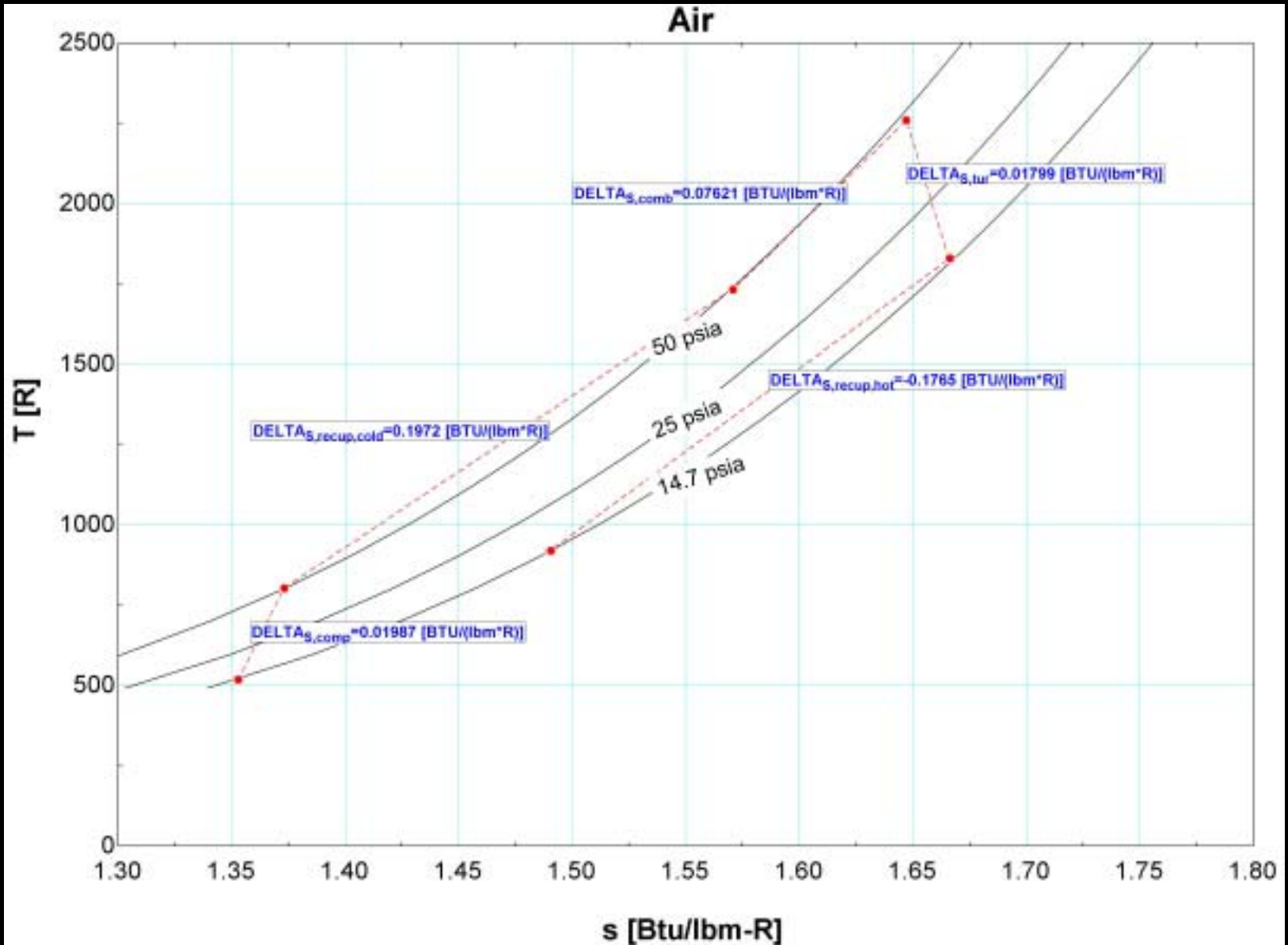


# Agreement Between First and Second Law Analysis



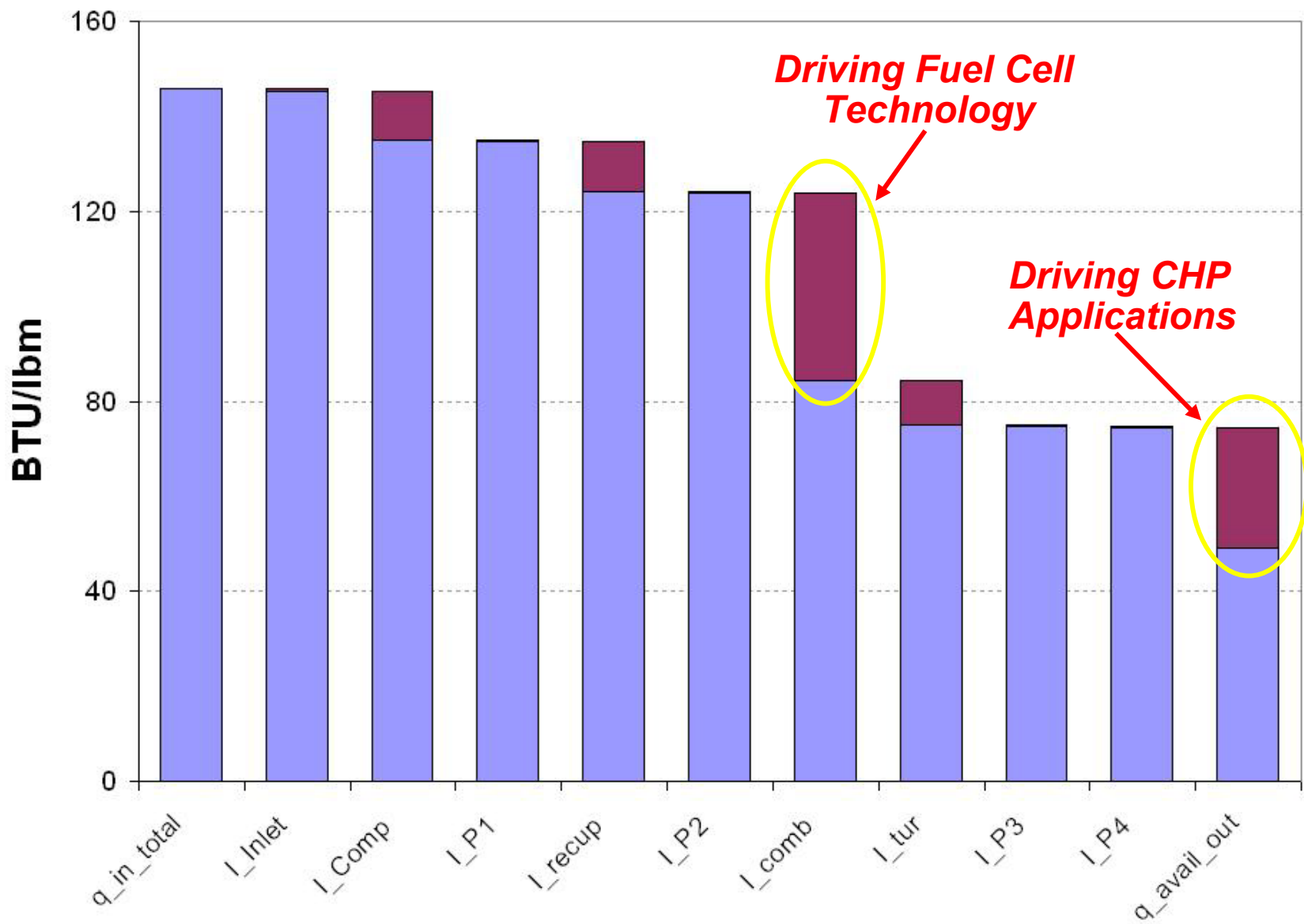


# Ideal Gas (Air) T-s Diagram





# Availability Stack-up





# Calculating Combustion Losses



$$\sum \left[ N_p \cdot (h_{fp}^o + h_p^o) \right] = \sum \left[ N_r \cdot (h_{fr}^o + h_r^o) \right] = (N \cdot h_f^o)_{\text{CH}_4}$$

$$\Phi = 0.6667 \quad T_{\text{prod}} = 1784\text{K}$$

$$\Delta S_{\text{surr}} = 0$$

$$S_{\text{gen}} = \Delta S_{\text{sys}} = S_{\text{prod}} - S_{\text{react}} = \sum N_p \cdot S_p - \sum N_r \cdot S_r$$

$$S_x = N_x \cdot s_x(T, P_x) = N_x \cdot \left[ s_x^o(T, P_o) - R \cdot \ln(y_x \cdot P_{\text{total}}) \right]$$

$$I = T_o \cdot S_{\text{gen}}$$

$$I = 288350 \frac{\text{kJ}}{\text{kmol} \cdot \text{K}}$$





# Calculating Combustion Losses



$$\frac{P_{\text{sat}}}{P_{\text{ref}}} \cdot (4.76 N_{\text{O}_2} + N_v) = N_v \quad N_v = 0.463 \text{ kmol} \quad N_l = 1.537 \text{ kmol}$$

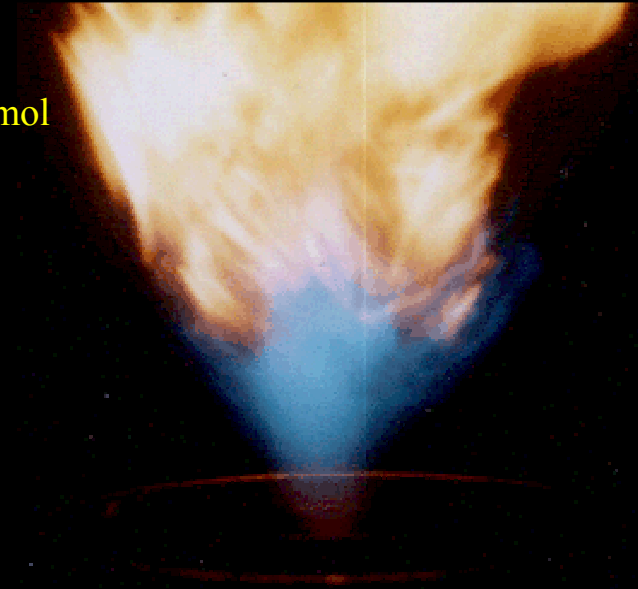
$$Q = \sum [N_p \cdot (h_{\text{fp}}^{\circ} + h_p^{\circ})] - \sum [N_r \cdot (h_{\text{fr}}^{\circ} + h_r^{\circ})] \quad T_{\text{prod}} = T_o$$

$$Q = -866686 \frac{\text{kJ}}{\text{kmol}}$$

$$\Delta S_{\text{sys}} = S_{\text{prod}} - S_{\text{react}} \quad S = \sum N_x \cdot \left[ s_x^{\circ}(T_o, P_o) - R \cdot \ln(y_x \cdot P_{\text{total}}) \right]$$

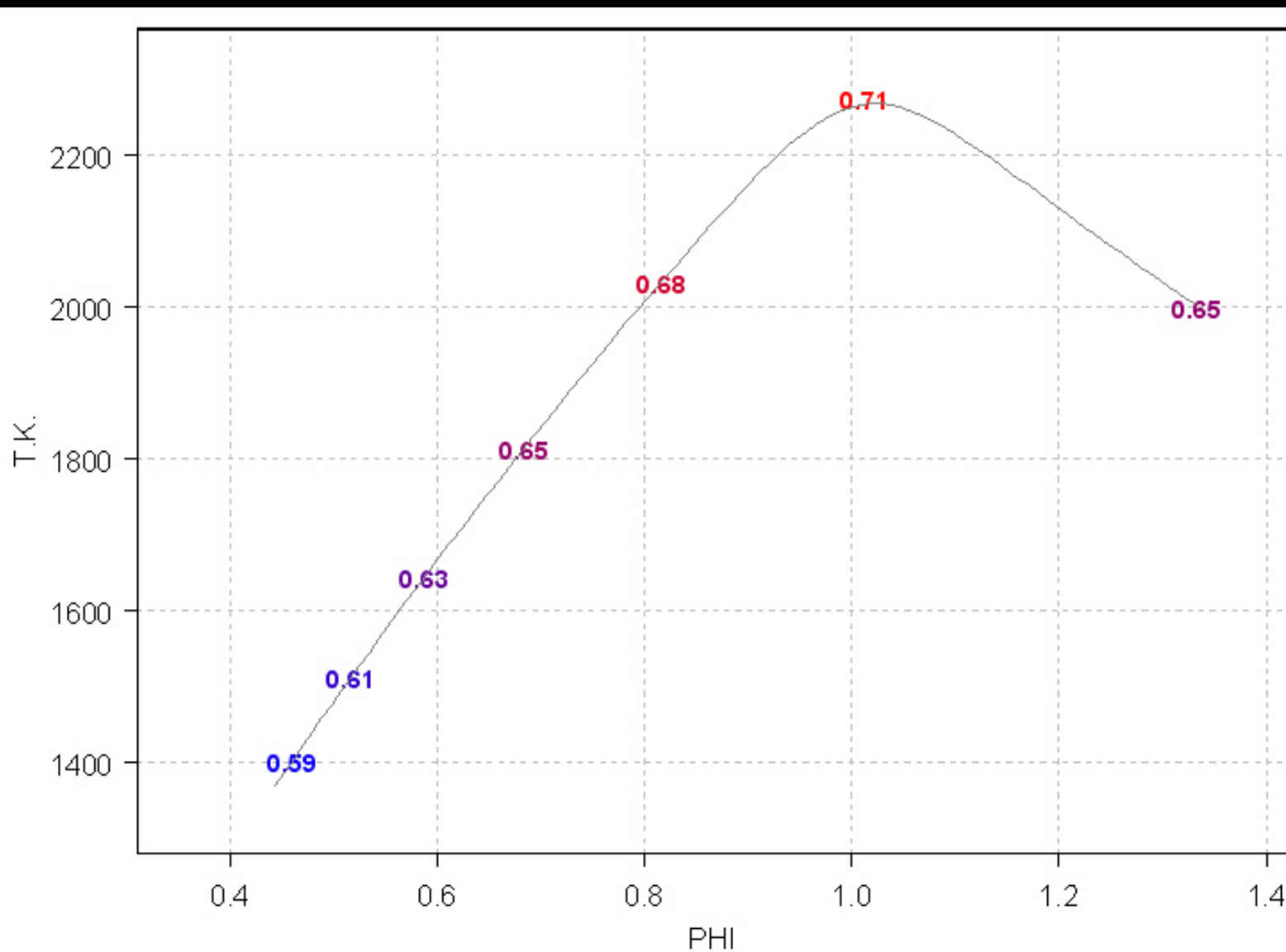
$$\Delta S_{\text{surr}} = \frac{-Q_{\text{surr}}}{T_o} \quad S_{\text{gen}} = \Delta S_{\text{sys}} + \Delta S_{\text{surr}} = 2769 \frac{\text{kJ}}{\text{kmol}_{\text{CH}_4}}$$

$$I = T_o \cdot S_{\text{gen}} = 825268 \frac{\text{kJ}}{\text{kmol}_{\text{CH}_4}} \quad \eta_s = \frac{I - W_{\text{irr}}}{I} = \frac{825268 - 288350}{825268} = 0.65$$





# Quantifying Combustion Losses







# Second Law Summary

- Determines the Maximum Work Potential of Any Thermal/Fluid System
- Provides Greater Insight To System Losses Than A First Law Analysis Alone
- A Powerful Tool To Optimize Thermal Efficiency
- Is No More Difficult To Understand or Implement Than First Law Analysis

